## PENTAIR

## JUNG PUMPEN

Calculation of sewage disposal units and pumping stations


## JUNG PUMPEN

CALCULATION OF SEWAGE DISPOSAL UNITS AND PUMPING STATIONS

## CONTENT

1. Introduction ..... 4
1.1 Overview of norms ..... 4
2. Pumping medium ..... 4
2.1 Waste water drainage $Q_{w w}$ ..... 5
2.2 Rainwater drainage $Q_{R}$ ..... 7
2.3 Domestic waste water $Q_{h}$. ..... 8
3. Pumping distance ..... 10
3.1 Pipe diameter ..... 10
3.2 Pipe friction losses $\mathrm{H}_{\mathrm{vL}}$ ..... 10
3.3 Losses $\mathrm{H}_{\mathrm{vE}}$ in fixtures, fittings and mouldings ..... 13
3.4 Static head $\mathrm{H}_{\text {geo }}$ ..... 16
3.5 Manometric head $\mathrm{H}_{\text {man }}$ ..... 16
3.6 Flow velocity v ..... 18
4. Pumping unit ..... 19
4.1 Single or duplex system ..... 19
4.2 Parallel connection of pumps ..... 19
4.3 Series connection of pumps ..... 20
4.4 Pressure pipe volume $V_{D}$ ..... 21
4.5 Switching period $\mathrm{T}_{\mathrm{Sp}}$ ..... 21
4.6 Pump volume $\mathrm{V}_{\mathrm{p}}$ ..... 21
4.7 Hysteresis $h_{p}$ ..... 22
4.8 Sump volume $\mathrm{V}_{\text {su }}$, switch-off level $\mathrm{h}_{\text {Aus }}$ ..... 23
5. Calculation examples ..... 24
5.1 Calculation example 1 ..... 24
5.2 Calculation example 2 ..... 28
6. Pump dimensioning support ..... 33
7. Design help for control technology ..... 34
8. Backpressure level ..... 35
9. Formula symbols used ..... 36
10. PEHD pressure pipes ..... 38

## 1. INTRODUCTION

The dimensioning of pumps and pressure pipes has to be done step by step. The four most important criteria are

- WHAT kind of medium? Pumping medium
- HOW MUCH volume? Pumping volume
- WHERETO, how far, how high? Pumping distance
- WHEREWITH is pumping supposed to take place? Pumping unit

For a better understanding of the symbols used in the formulas you will find a summary of the symbols used at the end of the text. Under 5. Calculation examples you will find two examples of typical application situations that will help you to easily orientate with your application situation.

### 1.1 OVERVIEW OF NORMS



Source: DIN 1986-100-chart1

## 2. PUMPING MEDIUM

On principle a distinction can be drawn between
$\begin{array}{ll}\text { - faeces free waste water } & \text { (grey water) and } \\ \text { - waste water containing faeces } & \text { (black water). }\end{array}$
However, there are further regulations that may have to be considered. Further information regarding your particular situation will be provide by the government safety organisation, the inspectorate division, local associations for technical inspections e.g. the German TÜV or the department of planning and building inspection.

### 2.1 WASTE WATER DRAINAGE $Q_{w w}$

Authoritative for the dimensioning is the quantity of waste water $Q_{w w}$ to be expected according to EN 12056-2, which is determined taking into account the concurrency from the sum of design units (DU), wherein $K$ is the reference point for the drainage characteristic. It depends on the kind of building and results from the frequency of usage of the drainage objects.
$Q_{c}$ is the continuous flow rate which is not subjected to any observations of concurrency (e.g. grease separator).

|  | $Q_{w w}$ | [1/s] | = Waste water drainage | Formula for amount determination |
| :---: | :---: | :---: | :---: | :---: |
| $\mathrm{Q}_{\mathrm{ww}}=\mathrm{K} \cdot \mathrm{v} \overline{\sum(\mathrm{DU})}$ | K |  | = drainage characteristic |  |
|  | [ DU |  | = Sum of connection values |  |
|  | $\mathrm{Q}_{\text {tot }}$ | [1/s] | = Total waste water drainage |  |
| $Q_{\text {tot }}=Q_{\text {ww }}+Q_{c}$ | $Q_{w w}$ | [ $1 / \mathrm{s}$ ] | = Waste water drainage |  |
|  | $\mathrm{Q}_{\text {c }}$ | [//s] | = Continuous drainage |  |

The waste water drainage $Q_{w w}$ can be calculated from the sum DU (table 2) with the above shown formula while taking the respective drainage characteristic K (table 1) into account. Table 3 may be used as an alternative to the calculation. If the calculated waste water drainage $Q_{w w}$ is smaller than the largest connection value of a single drainage object, the latter is authoritative (threshold value).

Tabelle 1: Typical drainage characteristics (K)

| Kind of building | K |
| :--- | :---: |
| irregular use, e. g. in residential buildings, guesthouses, offices | 0,5 |
| regular use, e. g. in hospitals, schools, restaurants, hotels | 0,7 |
| frequent use, e. g. in public restrooms and/or showers | 1,0 |
| special use, e. g. laboratories | 1,2 |

Source: EN 12056-2:2000, table 3

Table 2: Connection values DU

| Drainage object | $\begin{array}{\|c} \hline \text { System I } \\ D U \\ {[\mathrm{l} / \mathrm{s}]} \end{array}$ | $\begin{gathered} \hline \text { System } \\ \text { II } \\ \text { DU } \\ {[1 / \mathrm{s}]} \end{gathered}$ |
| :---: | :---: | :---: |
| sink, bidet | 0,5 | 0,3 |
| shower without stopper | 0,6 | 0,4 |
| shower with stopper | 0,8 | 0,5 |
| single urinal with tank | 0,8 | 0,5 |
| urinal with pressure flushing | 0,5 | 0,3 |
| floor standing urinal | 0,2* | 0,2* |
| tub | 0,8 | 0,6 |
| kitchen sink | 0,8 | 0,6 |
| dishwasher (household) | 0,8 | 0,6 |
| washing machine up to 6 kg | 0,8 | 0,6 |
| washing machine up to 12 kg | 1,5 | 1,2 |
| toilet with 4,0 l tank | ** | 1,8 |
| toilet with 6,0 I tank | 2,0 | 1,8 |
| toilet with 7,5 I tank | 2,0 | 1,8 |
| toilet with 9,0 I tank | 2,5 | 2,0 |
| floor drain DN 50 | 0,8 | 0,9 |
| floor drain DN 70 | 1,5 | 0,9 |
| floor drain DN 100 | 2,0 | 1,2 |
| * per person <br> ** not approved |  |  |

System I: Single unit with partially filled pressure pipes (filling ratio 0,5 or $50 \%$ respectively)

System II: Single unit with pressure pipes of smaller dimension (filling ratio of 0,7 or $70 \%$ respectively)

In accordance with nationaly regulations system I is used in Germany. Upon usage of water saving toilets system II may be applied.

[^0]Upon usage of water saving toilets the requirements for system type II in accordance with EN 12056-2 have to be observed as well:
The connected load for a toilet with $4,0 \mathrm{l}$ to $4,5 \mathrm{l}$ flushing has to be $\mathrm{DU}=1,8 \mathrm{l} / \mathrm{s}$
(Source: DIN 1986-100, 8.3.2.1)

Table 3: Conversion chart $\sum \mathrm{DU}$ in $\mathrm{Q}_{\mathrm{ww}}[\mathrm{L} / \mathrm{s}]$


## Example

| Drainage object | Quantity |  | DU | $\Sigma$ DU |
| :--- | :---: | :---: | :---: | :---: |
| hand wash basin | 2 | $\bullet$ | 0,5 | 1,0 |
| washing machine up to 6 kg | 1 | $\bullet$ | 0,8 | 0,8 |
| floor drain DN 100 | 1 | $\bullet$ | 2,0 | 2,0 |
| Toilet with $7,5 \mathrm{I}$ tank | 2 | $\bullet$ | 2,0 | 4,0 |
| tub | 2 | $\bullet$ | 0,8 | 1,6 |
| shower without stopper | 1 | $\bullet$ | 0,6 | 0,6 |
| $\Sigma$ DU |  |  | 10,0 |  |

When to the conduit with $\sum D U=10$ an additional conduit with $\sum D U=15$ is connected ( $K=0,5, \mathrm{e}$. g. housebuilding), the new sum of DU is then $10+15=25$.
Therewith the drainage of the continuing conduit is

$$
Q_{w w}=0,5 \cdot \sqrt{25}=2,5 \mathrm{l} / \mathrm{s}
$$

### 2.2 RAINWATER DRAINAGE $\mathbf{Q}_{\mathrm{R}}$

The amounts of rainfall are attributable to climate and vary greatly from region to region.
The occurring rainfalls are grouped depending on their respective frequency by:
$r_{5 / 2} \quad$ five minute rain which statistically has to be expected once in 2 years
$r_{5 / 100}$ five minute rain which statistically has to be expected once in 100 years
In DIN 1986-100 (appendix A, table A.1) the values for several German cities are exemplarily listed. The values differ from $r_{5 / 2}=200$ to $250 \mathrm{l} / \mathrm{s} \mathrm{N} \mathrm{ha)} \mathrm{or} \mathrm{r}_{5 / 100}=>800 \mathrm{l} / \mathrm{s} \cdot$ hal respectively. [1 ha $\left.=10.000 \mathrm{~m}^{2}\right]$ Indications regarding rainfalls are to be requested from the local authorities or alternatively from the Meteorological Service in your country. Guiding values are given in the DIN 1986-100 appendix A. If no values are available, $\mathrm{r}_{\mathrm{T}(\mathrm{n})}=200 \mathrm{l} /(\mathrm{s} \cdot \mathrm{ha})$ should be assumed.

Pipesystems and the respective component parts of rain drainage units are to be dimensioned for an average rainfall for economical reasons and in order to ensure self-cleaning capacity. The rain taken as calculation basis is, within the domain of DIN 1986-100, an idealised rainfall (block rainfall) with a constant rain intensity for 5 minutes. The respective annual factor ( $T_{n}$ ) to be used for the assessment situation is determined by the objective. Rainfalls above the calculation basis $\left(r_{5 / 2}\right)$ are to be expected in the planning.

| $Q R=r_{(D, T)} \cdot C \cdot A$ | $\frac{1}{10000}$ | $Q_{R}$ | [1/s] | = Rain water drainage |
| :---: | :---: | :---: | :---: | :---: |
|  |  | $\mathrm{r}_{(0, T)}$ | [l//s - ha] | = Assessment rainfall |
|  |  | C |  | = Run-off coefficient |
|  |  | A | [m²] | = Precipitation area <br> $=\left(1 \mathrm{ha}=10000 \mathrm{~m}^{2}\right)$ |

Tabelle 4: Abflussbeiwerte C zur Ermittlung des Regenwasserabflusses $\mathrm{Q}_{\mathrm{R}}$ DIN 1986-100

| No. | Kind of area | Rund-off coefficient C |
| :---: | :--- | :---: |
| 1 | Areas non-permeable to water, e.g. | 1,0 |
|  | - roofs > $3^{\circ}$ slope | 1,0 |
|  | - concrete areas | 1,0 |
|  | - ramps | 1,0 |
|  | - fixed areas with gap sealing | 1,0 |
|  | - bituminous layer | 1,0 |
|  | - pavement with joint filling | 1,0 |
|  | - roofs $\leqslant 3^{\circ}$ slope | 1,0 |
|  | - gravel roofs | 0,5 |
|  | - sodded roofs | 0,3 |
|  | - for intense soddings | 0,3 |
|  | - for extensive soddings beginning with 10 cm build-up gauge | 0,5 |
| 2 | - for extensive soddings with less than 10 cm build-up gauge |  |
|  | Partially permeable and poorly draining expanses, e. g. | 0,7 |
|  | - concrete paving placed in sand or slag, expanses with tiles | 0,6 |
|  | - expanses with paving, with >15\% of joints e. g. $10 \mathrm{~cm} \times 10 \mathrm{~cm}$ and smaller | 0,5 |
|  | - water bound areas | 0,3 |
|  | - partially surfaced playgrounds | 0,6 |
|  | - training grounds with drainage | 0,4 |
|  | - plastic expanses, artificial lawn | 0,3 |
|  | - barn floors |  |
| 3 | - lawns |  |
|  | Water-permeable expanses without or with insignificant drainage, e. g. |  |
|  | - parks and vegetation areas |  |
|  | - Gravel and slag floors, cobble |  |
|  | also with surfaced expanse parts such as | 0,0 |

[^1]
### 2.3 DOMESTIC WASTE WATER $\mathbf{Q}_{H}$

For the dimensioning of larger pumping systems with for example entire streets of houses or housing sites connected to them, it is not the EN 12056 (gravity drainage inside of buildings) that is accessed but the EN 752 (drainage systems outside of buildings) or respectively the ATV A 118 (hydraulic calculation and proof of drainage systems).

This ATV guideline describes the so called ,.dry weather discharge" $Q_{t}$. It includes the domestic waste water discharge $Q_{h}$, the commercial waste water discharge $Q_{g}$ and the infiltration water discharge $Q_{f}$ without rainwater.
$Q_{t}=Q_{h}+Q_{g}+Q_{f}$
Infiltration water can come from intruding ground water, from unauthorised connections or from discharged surface water, e. g. through untight duct covers. The infiltration water supplement should be $100 \%$ for the calculation of the waste water ducts. For mixed systems the infiltration water supplement can usually be neglected.

The calculation parameters specified here are used when, e.g., complete residential areas, villages etc. are supposed to be connected to a sewage system or a pumping station. For the calculation of gravity drainage within buildings the EN 12056 applies and for drainage systems outside of buildings the EN 752 (see above) applies.

The domestic waste water discharge Qh is significantly determined by the water consumption of the population. It is influenced by population density, structure, different ways of living, home culture and living standards

## The residential densities are between:

$20 \mathrm{E} /$ ha (rural areas, low-density development) and $300 \mathrm{E} /$ ha (city)
The average daily water consumption of the population including small businesses is between 80 and $200 \mathrm{l} /(\mathrm{E} \cdot \mathrm{d}$ ).
Recommendation: For the calculation of the future waste water drainage the prognoses of water consumption of the local water utility are to be taken as a basis.
However, for the purpose of the calculation the waste water factor should not fall below $150 \mathrm{l} /(\mathrm{E} \cdot \mathrm{d})$.
Daily fluctuations in the specific peak drains have to be taken into account. The hourly peak drain $[\mathrm{m} 3 / \mathrm{h}]$ is between $1 / 8$ (rural areas) and $1 / 16$ (large city) of the day value [m $3 / \mathrm{d}$ ].

Specific domestic waste water drainage
$\mathrm{q}_{\mathrm{h}}=0,005 \mathrm{l} /(\mathrm{s} \cdot \mathrm{E}) \quad$ or
$\mathrm{q}_{\mathrm{h}}=5,0 \mathrm{l} /(\mathrm{s} \cdot 1000 \mathrm{E})$

|  |  | $\mathrm{Q}_{\mathrm{h}}$ | [1/s] | = domestic waste water volume |
| :---: | :---: | :---: | :---: | :---: |
|  | $\mathrm{qh} \cdot \mathrm{ED} \cdot \mathrm{A}_{\mathrm{E}, \mathrm{k}, 1}$ | $\mathrm{q}_{\mathrm{h}}$ | [51/(s.1000 E)] | = specific domestic waste water drainage |
|  | 1000 | $A_{E, k, 1}$ | [ha] | = expanse of the residential area covered by the sewage system |
|  |  | ED | [E/ha] | = residential density within the draw area |

Example for 20000 inhabitants [E]

## Simplified calculation

$Q_{h}=q_{h} \cdot E$
$Q_{h}=0,005 \mathrm{l} /(\mathrm{s} \cdot \mathrm{E}) \cdot 20000 \mathrm{E}=100 \mathrm{l} / \mathrm{s}$
The quantities of inflow vary depending on the kind of connected area and time of day. An overview can be found in the following charts.

Chart 1: Inflow hydrographs (ATV A 134 - illustration 3 and 4)



## 3. PUMPING DISTANCE

### 3.1 PIPE DIAMETER

When the amount of inflow is determined, the conduit needs to be dimensioned. For the transport of waste water it applies in sewage technology that the minimum flow velocity upon transport may not be less than $\mathrm{v}_{\text {min }}=0,7 \mathrm{~m} / \mathrm{s}$ in order to avoid residues in the conduits. On the other hand it should not exceed $\mathrm{v}_{\max }=2,3 \mathrm{~m} / \mathrm{s}$ (EN 12056-4) in order to avoid flap impacts and water hammers as well as unnecessary waste of energy due to friction loss.

Therefore the ideal flow velocity has to be approximately between $\mathrm{v}_{\text {min }}=0,7$ and approx. $1,0 \mathrm{~m} / \mathrm{s}$. The conduit is selected in accordance with several criteria. Faeces free waste water (grey water) can be transported in pressure lines with a minimum diameter of DN 32 .

For waste waters containing faeces (black water) pressure lines with a minimum diameter of DN 80 are required in accordance with ATV guidelines or EN 12056 respectively, unless the pump is equipped with an adequate cutting device (e. g. MultiCut). In the EN 12056 the minimum diameter for pressure lines in site drainage with attached pumps with cutting units is set to DN 32 . If a sewage disposal unit for the disposal of a single toilet (e.g. WCfix) is used, the pressure line can be run in DN 25 . If the tubing diameter has not yet been defined, it is chosen in such a way that a minimum flow velocity of $\mathrm{v}_{\text {min }}>0,7 \mathrm{~m} / \mathrm{s}$ is maintained.

Now two cases are conceivable:
Case A: The volume to be pumped off is larger or equal to the volume that is required to reach the minimum flow velocity $\mathrm{v}_{\text {min }}$ in the fittings and the conduit.

Case B: The volume to be pumped off is smaller than the volume required to reach the minimum flow velocity. (Typical in single residence disposal and backpressure protection). In this case the minimum volume $Q$ to be transported is set to the volume which is required in order to reach the minimum flow velocity $\mathrm{v}_{\text {min }}$.

$Q=V_{D / m} \cdot V_{\text {min }} \quad$| $Q$ | $[\mathrm{l} / \mathrm{s}]$ | $=$ capacity |
| :--- | :--- | :--- |
| $\mathrm{V}_{\mathrm{D} / \mathrm{m}}$ | $[\mathrm{l} / \mathrm{m}]$ | $=$ volume of the pressure line/meter (see table 6 and 7 ) |
| $\mathrm{V}_{\text {min }}$ | $[\mathrm{m} / \mathrm{s}]$ | $=$ minimum flow velocity (generally $0,7 \mathrm{~m} / \mathrm{s}$ ) |

The actual volume of waste water is here only used to maybe later determine the energy costs.

### 3.2 PIPE FRICTION LOSSES Hy

The flowing of the pumping medium through the tubing results in friction losses. These losses depend on flow velocity, diameter and roughness of the pipework, viscosity of the pumping medium, number and kind of fixtures and the length of the tubing.

The smaller the diameter, the higher the flow velocity needs to be in order to pump the same volume through the pipe. The higher the flow velocity, the higher the friction losses. They rise by square in comparison to the flow velocity - this means a doubling of the flow velocity yields a quadruplicate of the friction losses.

Another factor is the operational roughness $k_{b}$ of the interior wall of the tubing. It can be between $0,1 \mathrm{~mm}$ and several millimetres. The determining factors are material and condition of the tubing.

If no specific defaults are given a standard factor of $k_{b}=0,25 \mathrm{~mm}$ should be used.
Table 5a or 5b or chart 2 are used to determine the pipe friction loss HvL depending on given or chosen tubing diameter and tubing length.
To do so the nomogramm is entered vertically with the relevant pumping volume until the vertical line intersects with the diagonal tubing line of the chosen diameter.
The other diagonals indicate the flow velocity of the volumes to be pumped in the chosen pipe.
When a horizontal line is now drawn starting from this point, the pipe friction loss for 100 m pipework can be read on the $y$-axis.
The chart is valid for an operational roughness of $\mathrm{k}_{\mathrm{b}}=0,25 \mathrm{~mm}$.

Chart 2: Pressure loss in pipes ( $\mathrm{kb}=0,25 \mathrm{~mm}$ ) v $=1,31 \mathrm{~mm} 2 / \mathrm{s}$ (water $10^{\circ} \mathrm{C}$ )


A linear conversion of the found $\mathrm{H}_{\mathrm{vL} 100}$ value may be performed for the existing conduit length.

1. Example: $d_{i}$ and $Q$ known, $H_{v L}$ searched
$Q=25 \mathrm{~m}^{3} / \mathrm{h}$
cuts DN 100 bei $H_{\text {vL100 }}=1,1 \mathrm{~m}$
$v=0,9 \mathrm{~m} / \mathrm{s}$

This means for a pressure line of 350 m :
$H_{v L}=\frac{1,1 \mathrm{~m}}{100 \mathrm{~m}} \cdot 350 \mathrm{~m}=\underline{\underline{3,85 \mathrm{~m}}}$
2. Example: $Q$ known, $H v L$ and di searched

Q $\quad=30 \mathrm{~m}^{3} / \mathrm{h} \quad$ cuts DN 100 between $v=0,7 \mathrm{~m} / \mathrm{s}$ and $2,3 \mathrm{~m} / \mathrm{s}$
chosen: $D N 100$, since $v L 0,7 \mathrm{~m} / \mathrm{s}$ and equals $H_{v L 100}=1,6 \mathrm{~m}$
$v=1,1 \mathrm{~m} / \mathrm{s}$

This means for a pressure line of 160 m :
$H_{v L}=\frac{1,6 \mathrm{~m}}{100 \mathrm{~m}} \cdot 160 \mathrm{~m}=\underline{\underline{2,56 \mathrm{~m}}}$
For the estimate assessment of pipe friction losses table 5 a or 5 b may also be used.
Table 5a: Pipe friction losses $\mathrm{H}_{\mathrm{vL} 100}$ per $\mathbf{1 0 0} \mathbf{m}$ conduit length

| Flow rate in $\mathrm{m}^{3} / \mathrm{h}$ | Inner diameter in mm |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | 25 | 32 | 40 | 50 | 65 | 80 | 100 | 125 | 150 | 200 | 250 |
| 2 | 11,0 | 2,9 | 0,9 | 0,3 |  |  |  |  |  |  |  |
| 4 | 43,0 | 11,2 | 3,6 | 1,1 | 0,3 |  |  |  |  |  |  |
| 6 | 95,0 | 26,0 | 7,7 | 2,4 | 0,6 | 0,2 |  |  |  |  |  |
| 8 |  |  | 13,5 | 4,2 | 1,1 | 0,4 |  |  |  |  |  |
| 10 |  |  | 21,0 | 6,5 | 1,7 | 0,6 | 0,2 |  |  | ${ }_{00}$ |  |
| 15 |  |  |  | 14,5 | 3,7 | 1,3 | 0,4 | 0,1 |  |  |  |
| 20 |  |  |  | 25,5 | 6,5 | 2,2 | 0,7 | 0,2 |  |  |  |
| 25 |  |  |  | 39,6 | 10,0 | 3,4 | 1,1 | 0,3 |  |  |  |
| 30 |  |  |  |  | 14,3 | 4,9 | 1,5 | 0,5 |  |  |  |
| 35 |  |  |  |  | 19,4 | 6,6 | 2,1 | 0,7 | 0,3 |  |  |
| 40 |  |  |  |  | 25,3 | 8,5 | 2,7 | 0,9 | 0,3 |  |  |
| 45 |  |  |  |  | 31,9 | 10,8 | 3,4 | 1,1 | 0,4 |  |  |
| 50 |  |  |  |  |  | 13,2 | 4,1 | 1,3 | 0,5 |  |  |
| 55 |  |  |  |  |  | 16,0 | 5,0 | 1,6 | 0,6 |  |  |
| 60 |  |  |  |  |  | 19,0 | 5,9 | 1,9 | 0,7 |  |  |
| 70 |  |  |  |  |  | 25,8 | 8,0 | 2,5 | 1,0 | 0,2 |  |
| 80 |  |  |  |  |  | 33,6 | 10,4 | 3,3 | 1,3 | 0,3 |  |
| 100 |  |  |  |  |  |  | 16,2 | 5,1 | 2,0 | 0,5 |  |
| 150 |  |  |  |  |  |  |  | 11,3 | 4,4 | 1,0 | 0,3 |
| 200 |  |  |  |  |  |  |  | 19,9 | 7,7 | 1,7 | 0,5 |
| 300 |  |  |  |  |  |  |  |  | 17,2 | 3,8 | 1,2 |
| 400 |  |  |  |  |  |  |  |  | 30,4 | 6,8 | 2,1 |

absolute roughness $\mathrm{k}_{\mathrm{b}}=0,25 \mathrm{~mm}$ kinematic viscosity $v=1,31 \mathrm{~mm}^{2} / \mathrm{s}$ (water $10^{\circ} \mathrm{C}$ )

Table 5b: Friction losses $\mathrm{H}_{\mathrm{vL} 100}$ for pressure pipework as per DIN 14811

| Flow rate <br> in $\mathrm{m}^{3} / \mathrm{h}$ | Losses in mWS per length of 100 m |  |  |
| :---: | :---: | :---: | :---: |
| 6 | C 42 | C 52 | B 75 |
| 9 | 4 | 2 | - |
| 12 | 9 | 3 | - |
| 15 | 15 | 5 | 1 |
| 18 | 22 | 8 | 1,5 |
| 21 | 30 | 11 | 2 |
| 24 | 41 | 15 | 3 |
| 30 | 53 | 20 | 4 |
| 36 | - | 26 | 5 |
| 48 | - | 40 | 8 |
| 60 | - | 65 | 14 |
| 72 | - | - | 22 |
| 96 | - | - | 30 |

### 3.3 LOSSES $\mathrm{H}_{\mathrm{VE}}$ IN MOUNTING PARTS, VALVES AND FITTINGS

Additional values are the flow losses in the installation. In order to determine the resistance coefficients for valves and fittings it is sufficiently precise to take the respective $\zeta$-values from table 6 .

Table 6: Drag coefficient for valve and shaped parts

| Mounting part | DN | $\zeta$-values |
| :---: | :---: | :---: |
| GR 35/40 foot + claw |  | 1,30 |
| GR 50 foot + claw |  | 1,00 |
| GR-system | 65 | 0,25 |
| GR-system | 80-150 | 0,22 |
| Bend $45^{\circ}, \quad R / D=2,5$ |  | 0,20 |
| Elbow 45 ${ }^{\circ}$, $\quad R / D=1,0$ |  | 0,35 |
| Bend $90^{\circ}$, $\quad R / D=2,5$ |  | 0,35 |
| Elbow $90^{\circ}$, $\quad R / D=1,0$ |  | 0,50 |
| Flat slide | 32 | 0,50 |
| Flat slide | 40 | 0,46 |
| Flat slide | 50 | 0,42 |
| Flat slide | 80 | 0,36 |
| Flat slide | 100 | 0,34 |
| Flat slide | 150 | 0,30 |
| T-piece | 80 | 1,30 |
| T-piece | 100 | 1,30 |
| T-piece | 150 | 1,30 |
| T-piece | 200 | 1,30 |


|  |  | $B$ |
| :--- | :---: | :---: |
| Extension | 50/ $40=1,25$ | $8^{\circ}$ |
| Extension | $100 / 80=1,25$ | $8^{\circ}$ |
| Extension | $150 / 100=1,5$ | $8^{\circ}$ |
| Extension | $200 / 150=1,33$ | $8^{\circ}$ |
| Extension | $50 / 40=1,25$ | $10^{\circ}$ |
| Extension | $100 / 80=1,25$ | $10^{\circ}$ |
| Extension | $150 / 100=1,5$ | $10^{\circ}$ |
| Extension | $200 / 150=1,33$ | $10^{\circ}$ |
| Extension | $50 / 40=1,25$ | $18^{\circ}$ |
| Extension | $100 / 80=1,25$ | $18^{\circ}$ |
| Extension | $150 / 100=1,5$ | 0,12 |
| Extension | $200 / 150=1,33$ | $18^{\circ}$ |
| Free outflow |  | $18^{\circ}$ |

The $\zeta$-values of the reflux valves can be taken from chart 3 depending on the capacity Q .
The found $\zeta$-values are added up ( $\Sigma \zeta$ ).
With the help of chart 4 the chart is entered starting at the top left with the capacity Q . When the chosen pipe diameter $d_{i}$ is intersected, the line has to be drawn straight down to the second part of the chart starting with this point. From there it is taken parallely to the lines of the flow velocity until the line with the sum of zeta values is intersected. Starting from here the line is again drawn straight down which yields the pressure loss factor $\mathrm{H}_{\mathrm{vE}}$.

Example: $Q=30 \mathrm{~m}^{3} / \mathrm{h}$

| Mounting parts |  |  |  | $\zeta$-complete |
| :--- | :---: | :---: | :---: | :---: |
| 1 valve DN 100 | 0,34 | 0,34 |  |  |
| 2 elbows $90^{\circ}$, DN 100 | 0,50 | 1,00 |  |  |
| 1 swing-type check valve R 100 G | 7,00 | 7,00 |  |  |
| weight in the middle |  | $\Sigma \zeta$ |  |  |
| Sum | 8,34 |  |  |  |

Chart 4 results in: $\mathrm{H}_{\mathrm{VE}}=0,45 \mathrm{~m}$
$H_{v E}$ is then added to $H_{v L}$ addiert: $H_{v}=H_{v L}+H_{v E}$

$8=$ R 80 G - weight in middle position
$9=$ R100 G - weight in outer position
$10=$ R100 G - weight in middle position
11= R150 G - weight in outer position
$12=$ R150 G - weight in middle position

Chart 4: Level of pressure loss for mounting parts and fittings


### 3.4 GEODETIC HEAD Hgeo

The geodetic head indicates the difference between switch-off point of the pump and the connection to the gravity sewer. The pump has to "lift" the pumping medium this far.


The geodetic head is a system constant that cannot be altered. Therefore it is also set in the Q-H-chart as a constant to which the other losses $\mathrm{H}_{\mathrm{v}}$ are added.


### 3.5 MANOMETRIC HEAD H ${ }_{\text {man }}$

The addition of $\mathrm{H}_{\mathrm{v}}$ and $\mathrm{H}_{\text {geo }}$ yields the manometric head $\mathrm{H}_{\text {man }}$, necessary for selecting a pump.
$H_{\text {man }}=H_{v}+H_{\text {geo }}$
With this calculated factor and with the required pumping volume a suitable pump for the case of use is chosen. The characteristic of the pump has to be above or on this desired operation point (1).


At this point, however it can only be stated that the pump is able to pump the occurring waste water volume. A statement regarding the actual operation point is not yet possible. To do so it is necessary to assess the pipework or unit characteristic.

When one assumes several different quantities $Q$ and determines the factors $H v$ for these, one finds several points that can be entered into the Q-H-chart. The connection of these points equals the tubing or unit characteristic.


In order to now determine the actual operation point of the pump the intersection point of unit characteristic and pump characteristic has to be found. This is the actual operation point of the pump. It is both in capacity and manometric head higher than the desired operation point found for the pre-selection of the pump.


### 3.6 FLOW VELOCITY V

The volumes of the pressure pipe per meter VD/m are needed for the verification of the flow velocity. For short lengths of pipework it is sufficiently exact to work with the values of table 7.
$\mathrm{v}_{\text {min }} \leqslant \quad \mathrm{v}<\mathrm{v}_{\text {max }}$
$0,7 \mathrm{~m} / \mathrm{s} \leqslant \quad v \quad<\quad 2,3 \mathrm{~m} / \mathrm{s}$

Table 7

| DN | 25 | 32 | 40 | 50 | 65 | 80 | 100 | 125 | 150 | 200 | 250 | 300 |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{VD} / \mathrm{m}(\mathrm{I} / \mathrm{m})$ | 0,5 | 0,8 | 1,3 | 2 | 3,3 | 5 | 8 | 12,3 | 18 | 31 | 50 | 71 |

When larger distances have to be crossed it is necessary to calculate with the exact volumes or respectively diameters (table 8) because otherwise the derivations occurring in the loss factors will be too great.

The inner diameters vary in part greatly due to the different materials and their solidity. Among other things this is due to the fact that the outer diameters of the plastic pipes are the same while different wall thiknesses result in different inner diameters.

Example:
PVC-pipe DN 100, PN 10
PEHD-pipe DN 100, PN 10

| $110 \times 5,3 \mathrm{~mm}$ | $d_{i}=99,4 \mathrm{~mm}$ | $\mathrm{~V}_{\mathrm{D} / \mathrm{m}}=7,76 \mathrm{l} / \mathrm{m}$ |
| :--- | :--- | :--- |
| $110 \times 10 \mathrm{~mm}$ | $\mathrm{~d}_{\mathrm{i}}=90,0 \mathrm{~mm}$ | $\mathrm{~V}_{\mathrm{D} / \mathrm{m}}=6,36 \mathrm{l} / \mathrm{m}$ |

Table 8: gauge table of common pressure lines

|  | Grey cast iron pipes PN 16 DIN 28610 class K10 |  |  |  | PVC-pipes DIN 8061/8062 <br> PN 10 serial 4 |  |  |  | $\begin{gathered} \text { PEHD-pipes DIN } 8074 \\ \text { PN } 12,5 \text { - PE } 80 \text { - SDR } 11 \\ \hline \end{gathered}$ |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| DN | $\begin{gathered} \hline \mathrm{D} \times \mathrm{s} \\ {[\mathrm{~mm}]} \end{gathered}$ | $\begin{gathered} \mathrm{d}_{\mathrm{i}} \\ {[\mathrm{~mm}]} \end{gathered}$ | $\begin{aligned} & \mathrm{V}_{\mathrm{D} / \mathrm{m}} \\ & {[\mathrm{l} / \mathrm{m}]} \end{aligned}$ | $\begin{aligned} & \mathrm{Q}_{\text {min }} \\ & {[\mathrm{m} / \mathrm{s}]} \end{aligned}$ | $\begin{gathered} \hline \mathrm{D} \times \mathrm{s} \\ {[\mathrm{~mm}]} \end{gathered}$ | $\begin{gathered} \mathrm{d}_{\mathrm{i}} \\ {[\mathrm{~mm}]} \end{gathered}$ | $\begin{aligned} & \mathrm{V}_{\mathrm{D} / \mathrm{m}} \\ & {[\mathrm{l} / \mathrm{m}]} \end{aligned}$ | $\begin{aligned} & \mathrm{Q}_{\min } \\ & {[\mathrm{m} / \mathrm{s}]} \end{aligned}$ | $\begin{gathered} \hline \mathrm{D} \times \mathrm{s} \\ {[\mathrm{~mm}]} \end{gathered}$ | $\begin{gathered} \mathrm{d}_{\mathrm{i}} \\ {[\mathrm{~mm}]} \end{gathered}$ | $\begin{aligned} & \mathrm{V}_{\mathrm{D} / \mathrm{m}} \\ & {[\mathrm{l} / \mathrm{m}]} \end{aligned}$ | $\begin{aligned} & \mathrm{Q}_{\min } \\ & {[\mathrm{m} / \mathrm{s}]} \end{aligned}$ |
| 25 |  |  |  |  | $32 \times 1,5$ | 28,4 | 0,63 | 0,44 | $32 \times 2,9$ | 26,2 | 0,54 | 0,38 |
| 32 |  |  |  |  | $40 \times 1,9$ | 36,2 | 1,03 | 0,72 | $40 \times 3,7$ | 32,6 | 0,83 | 0,58 |
| 40 |  |  |  |  | $50 \times 2,4$ | 45,2 | 1,60 | 1,12 | $50 \times 4,6$ | 40,8 | 1,31 | 0,92 |
| 50 |  |  |  |  | $63 \times 3,0$ | 57,0 | 2,55 | 1,79 | $63 \times 5,8$ | 51,4 | 2,07 | 1,45 |
| 65 |  |  |  |  | $75 \times 3,6$ | 67,8 | 3,61 | 2,53 | $75 \times 6,8$ | 61,4 | 2,96 | 2,07 |
| 80 | $98 \times 9,0$ | 80 | 5,03 | 3,52 | $90 \times 4,3$ | 81,4 | 5,20 | 3,64 | $90 \times 8,2$ | 73,6 | 4,25 | 2,98 |
| 100 | $118 \times 9,0$ | 100 | 7,85 | 5,50 | $110 \times 5,3$ | 99,4 | 7,76 | 5,43 | $110 \times 10,0$ | 90,0 | 6,36 | 4,45 |
|  |  |  |  |  | $125 \times 6,0$ | 113,0 | 10,03 | 7,02 |  |  |  |  |
|  |  |  |  |  |  |  |  |  | $125 \times 11,4$ | 102,2 | 8,20 | 5,74 |
| 125 | $144 \times 9,2$ | 125,6 | 12,39 | 8,67 | $140 \times 6,7$ | 126,6 | 12,59 | 8,81 | $140 \times 12,7$ | 114,6 | 10,31 | 7,22 |
| 150 | $170 \times 9,5$ | 151,0 | 17,91 | 12,54 | $160 \times 7,7$ | 144,6 | 16,42 | 11,50 | $160 \times 14,6$ | 130,8 | 13,44 | 9,41 |
|  |  |  |  |  |  |  |  |  | $180 \times 16,4$ | 147,2 | 17,02 | 11,91 |
|  |  |  |  |  | $180 \times 8,6$ | 162,8 | 20,82 | 14,57 | $200 \times 18,2$ | 163,6 | 21,02 | 14,71 |
|  |  |  |  |  | $200 \times 9,6$ | 180,8 | 25,67 | 17,97 |  |  |  |  |
| 200 | $222 \times 10,0$ | 202,0 | 32,05 | 22,43 | $225 \times 10,8$ | 203,4 | 32,49 | 22,75 | $225 \times 20,5$ | 184,0 | 26,59 | 18,61 |
|  |  |  |  |  | $250 \times 11,9$ | 226,2 | 40,19 | 28,13 | $250 \times 22,7$ | 204,6 | 32,88 | 23,02 |
| 250 | $274 \times 10,5$ | 253,0 | 50,27 | 35,19 | $280 \times 13,4$ | 253,2 | 50,35 | 35,25 | $280 \times 25,4$ | 229,2 | 41,26 | 28,88 |
|  |  |  |  |  | $315 \times 15,0$ | 285,0 | 63,79 | 44,66 | $315 \times 28,6$ | 257, | 52,20 | 36,54 |
| 300 | $326 \times 11,0$ | 304,0 | 72,58 | 50,81 | $355 \times 16,9$ | 321,2 | 81,03 | 56,72 | $355 \times 32,3$ | 290,6 | 66,33 | 46,43 |

$\mathrm{VL}=$ notation from pipe norms

## 4. PUMPING UNIT

### 4.1 SINGLE OR DUPLEX SYSTEM

Systems for overlookable single cases can be executed as single units with only one pump. In systems where waste water transport may not be interrupted a double unit must be installed (EN 12056-4). In this case the pumping capacity of one pump has to be chosen in such a way that it can pump the maximum quantity of possible waste water.

### 4.2 PARALLEL CONNECTION OF PUMPS

When two pumps for example in a duplex pumping system pump together into one pipeline, the total pumping volume is larger than with single use, but not twice as large. The reason therefore can be found in the loss factors which rise in squares to the flow velocity.
In order to determine the performance of the pumps the respective pump characteristics are graphically added. To this effect auxiliary lines can be entered into the Q-H-chart, from which the same amounts (a) are respectively deducted from left to right. These new points P are connected and yield the collective characteristic of pump 1 and 2.


The intersection points with the unit characteristic yield the operation points $Q_{1,2} / H_{1,2}$ for single use of the pumps 1 or 2 and the operation points $Q_{1+2} / H_{1+2}$ for parallel use of both pumps. The performance of the single pumps is found by drawing a horizontal line (n) from the intersection point of the unit characteristic with the combined pump characteristic.
The $x$-axis now shows that with equal pumps 1 and 2 each pump pumps approximately half of the total pumping quantity but clearly less than when working alone.


$324560-00$

### 4.3 SERIES CONNECTION OF PUMPS

When a greater pumping volume is reached by paralleling, higher pumping heads can be reached by series connection.
The way of graphic illustration and determination of the operation points is to be performed similar to the parallel connection. The difference is in not adding over the pumping volume but over the pumping head.


### 4.4 PRESSURE PIPE VOLUME V

In order to avoid that the waste water remains for long periods of time in the pressure line and therewith causes unpleasant odour at the transfer chamber it is sensible that the pipe volume is exchanged with each operating cycle of the pumps by the pump volume V p, provided that the length of the pipe allows this.
$V_{p} \geqslant V_{D}$ ?
When the exchange is not ensured, an appropriate pressure flushing unit has to be used in case of need.
The pipe volume per meter $\mathrm{V}_{\mathrm{D} / \mathrm{m}}$ can be taken from table 7 or respectively table 8 . The pipe volume VD results from the following formula:

$$
V_{D}=V_{D / m} \cdot L_{D} \quad V_{D / m}[l / m] \quad=\text { Volume of the pressure pipe per meter }
$$

$L_{D} \quad[m] \quad=$ Length of the pressure pipe

### 4.5 DURATION OF SWITCHING INTERVAL T ${ }_{\text {SP }}$

In order to avoid undue strain to the motors of the pump due to too frequent starting (chatter effect) there are minimum switch-on intervals for the duration of switching intervals TSp depending on the assumed motor performance P1.
With motors that are at least half way immersed into the sump well, longer switching intervals are also permissible.

Table 9: Duration of switching interval $\mathrm{T}_{\mathrm{sp}}$

| for motors up to $\mathrm{P} 1=4 \mathrm{~kW}$ (direkt starting) | $\mathrm{T}_{\mathrm{S}_{\mathrm{p}}}=120 \mathrm{~s}$ |
| :--- | :--- |
| for motors up to $\mathrm{P} 1=7,5 \mathrm{~kW}$ (star-delta starting) | $\mathrm{T}_{\mathrm{S}_{\mathrm{p}}}=144 \mathrm{~s}$ |
| for motors up to $\mathrm{P} 1=7,5 \mathrm{~kW}$ (star-delta starting) | $\mathrm{T}_{\mathrm{Sp}}=180 \mathrm{~s}$ |

### 4.6 PUMP VOLUME V ${ }_{P}$

For the selection of the right sump and for adjusting the level contactors it is necessary to determine the minimum pumping volume Vp.
The pumping volume is the volume between switch-on and switch-off point of the pump in the sump.
When the inlet volume $Q z$ does not fluctuate greatly the following formula can be used for calculating:

|  | $V_{p}=\frac{T_{s p} \cdot Q_{z} \cdot\left(Q_{p}-Q_{z}\right)}{}$$Q_{p}$ $[l]$ $=$ Pumping volume <br> $T_{S p}$ $[s]$ $=$ Duration of switching interval <br> $Q_{z}$ $[l / s]$ $=$ Inlet volume <br> $Q_{p}$ $[l / s]$ $=$ Pumping volume of the pump at operation point |
| :--- | :--- | :--- |

Upon greatly fluctuating inlet volumes, like for example in rainwater pumping stations - soft or hard rain - the maximum required pumping volume should be determined for the calculation of the pumping volume $V_{p}$.

It is reached when

$$
Q_{z}=\frac{Q_{p}}{2} \text { is entered into the formula. }
$$

### 4.7 HYSTERESIS H

The hysteresis $h_{p}$, i. e. the adjustment distance of the level contactors within the PKS, is taken from chart 13 by means of the determined minimum pumping volume $\mathrm{V}_{\mathrm{p}}$.
The hysteresis $h_{p}$ in chart 14 are valid for the standard water sump inlets.
In case of special versions, the curves need to be adjusted.

## Chart 13: hysteresis $h_{p}$ PKS-A/PKS-B/ PKS-D


(1) PKS 800-32
(4) PKS-01000-065/080
(2) PKS 800-032
(5) PKS-01500-040
(3) PKS-01000-40/040
(6) PKS-D1500-080/0100

Chart 14: hysteresis $\mathrm{h}_{\mathrm{p}}$ DKS/KS/KSS


| $\mathbf{1}$ | DKS | $1000-50 / D 50$ | $\mathbf{3}$ | KS | $1500-\mathrm{D} 180$ | $\mathbf{4}$ | KS | $2000-\mathrm{D} 100$ |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| $\mathbf{2}$ | KS | $1500-\mathrm{D} 150$ |  | KS | $1500-\mathrm{D} 100$ |  | KS | $2000-\mathrm{D} 150$ |

### 4.8 SUMP VOLUME V $_{\text {su }}$, SWITCH-OFF LEVEL $H_{\text {AUs }}$

The volume remaining in the sump $\mathrm{V}_{\text {su }}$ and the switch-off level $\mathrm{h}_{\text {Aus }}$ can be determined by means of the table below. It has to be verified if the switch-off level is larger or equals the factor which ensures that the spiral casing of the pump does not emerge (see dimensions of the chosen pump).
The banking level $h$ results from $h=h_{p}+h_{\text {Aus }}$
Table 10: level switch-off points $\mathrm{h}_{\text {Aus }}$ / sump volume $\mathrm{V}_{\mathrm{su}}$
Sumps for sewage pumps UAK/UFK

|  | PKS-A 800 |  | $\begin{gathered} \text { PKS-B } 800 \\ -D 32 \end{gathered}$ | $\begin{gathered} \text { PKS-D } 1000 \\ -40 / \\ D 40 \\ \hline \end{gathered}$ | PKS-D 1000 |  | PKS-A 1200 |  | PKS-D 1500 |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | -D 32 | -80 |  |  |  | $\begin{aligned} & \text {-D } 65 \\ & \text {-D } 80 \end{aligned}$ | $\begin{aligned} & -40 / \\ & D 40 \end{aligned}$ | $\begin{aligned} & -80 / \\ & \text { D } 80 \end{aligned}$ | $\begin{aligned} & -40 / \\ & D 40 \end{aligned}$ | $\begin{gathered} \text {-D 80/ } \\ \text { D } 100 \end{gathered}$ |
| $\mathrm{h}_{\text {Aus }}$ [mm] | 270 | 250 | 200 | 270 | 230 | 340 | 200 | 240 | 280 | 280 |
| $\mathrm{V}_{\text {su }}[1]$ | 89 | 129 | 45 | 118 | 90 | 170 | 480 | 515 | 225 | 295 |


|  | DKS 1000 | KS 1000 |  | KS 1500 | KS/KSS 1500 | KS 2000 | KSS 2000 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{aligned} & -50 / \\ & \text { D } 50 \end{aligned}$ | 65/80 | $\begin{gathered} \text {-D } 65 / \\ \text { D } 80 \end{gathered}$ | -D 50 | $\begin{gathered} \text {-D 80/ } \\ \text { D } 100 \end{gathered}$ | $\begin{gathered} \text {-D } 100 / \\ \text { D } 150 \end{gathered}$ | $\begin{gathered} \text {-D } 100 / \\ \text { D } 150 \end{gathered}$ |
| $\mathrm{h}_{\text {Aus }}$ [mm] | 250 | 200 | 250 | 250 | 250 | 300 | 350 |
| $\mathrm{V}_{\text {su }}[1]$ | 150 | 135 | 178 | 196 | 310 | 540 | 660 |

Sumps for drainage pumps U/US

|  | PKS-A 800 <br> $-40 / D 40$ |
| :--- | :---: |
| $\mathrm{~h}_{\text {Aus }}$ [mm] | 235 |
| $\mathrm{~V}_{\text {su }}[\mathrm{ll]}$ | 137 |

Upon selection of a provided sump attention has to be paid that the measurement h is not larger than the maximum banking level from sump bottom to approx. 100 mm below the lower edge of the inlet. Should this be the case the inlet has to be relocated or a respectively larger sump has to be selected.


## 5. CALCULATION EXAMPLES

### 5.1 CALCULATION EXAMPLE 1

## Sewage disposal unit for waste water containing faeces

## 1. Determination of volumes in accordance with DIN EN 12056-2

Installation location: boarding house
$\Rightarrow K=0,5 \mathrm{l} / \mathrm{s}$
Depending of the kind of object to be drained the concurrence of use of the connected drainage objects is determined by the factor K .
The following drainage objects are to be discharged (system I):

|  | connecting valves |  |
| :--- | ---: | ---: |
| name | DU | $\Sigma D U$ |
| 12 hand wash basins | 0,5 | 6,0 |
| 8 toilets (with 6 l tank) | 2,0 | 16,0 |
| 4 urinals | 0,5 | 2,0 |
| 2 floor drains DN 70 | 1,5 | 3,0 |
|  |  | $\Sigma$ | connected to it:

1 dishwasher
2 kitchen drains
2 sinks in the kitchen
The total volume therewith results at:

| $Q_{w w}$ | $\begin{aligned} & =K \\ & =0,5 \end{aligned}$ | - $\sqrt{\sum D U_{s}}+Q_{c}$ <br> - $\sqrt{27,0}+2,0 \mathrm{l} / \mathrm{s}$ | $Q_{w w}$ | [l/s] | = waste water discharge |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | K |  | = drainage characteristic |
|  |  |  | DU | [l/s] | = connecting valves |
| $Q_{w w}$ | $=4,60 \mathrm{l} / \mathrm{s}$ |  | $Q_{\text {c }}$ | [l/s] | = continuous discharge |

Note: When the determined waste water discharge $Q_{w w}$ is smaller than the largest connection value of a single drainage object, the latter is controlling!

## 2. Verification of the minimum flow velocity $\mathbf{v}_{\text {min }}$

The unit is supposed to be connected by way of an existing pressure line DN 100 with a length of $L=25 \mathrm{~m}$.
The operational roughness is $\mathrm{kb}=0,25 \mathrm{~mm}$. The minimum flow velocity in pressure lines is $\mathrm{v}_{\min }>0,7 \mathrm{~m} / \mathrm{s}$
The pressure line has a volume
$V_{D / m}=8 \mathrm{l} / \mathrm{m} \quad$ lsee table 7 - the values are approximated values for $d_{i}=D N$. With larger tubing lengths it is recommended to calculate with the actual inner diameters of the pipes from table 8 . These, however, vary greatly due to the different materials and the resulting wall thiknesses of the pipes.)

This means that a minimum capacity $Q$ of

$$
\begin{array}{rll}
\mathrm{Q} & =\mathrm{V}_{\mathrm{D} / \mathrm{m}} & \bullet 0,7 \mathrm{~m} / \mathrm{s} \\
& =8,0 \mathrm{l} / \mathrm{m} & \bullet 0,7 \mathrm{~m} / \mathrm{s} \\
\mathrm{Q} & =5,6 \mathrm{l} / \mathrm{s} &
\end{array}
$$

is here required.

Upon verifying if the occurring capacity is larger than the necessary one ( $Q w w \geqslant \mathbb{Q}$ ) it is determined that
$Q_{w w}<Q$
$4,60 \mathrm{l} / \mathrm{s}<5,60 \mathrm{l} / \mathrm{s}$
This means that the next steps of the calculations are not performed with the actually occurring volume of waste water but that the volume necessary for reaching the minimum flow velocity is applied.
$Q=5,60 \mathrm{l} / \mathrm{s}$ With the conversion factor 3,6 a conversion can be performed from unit [l/s] to [m3/h].
$Q=20,16 \mathrm{~m}^{3} / \mathrm{h} \approx 20 \mathrm{~m}^{3} / \mathrm{h}$

## 3. Determination of the tubing friction losses of the tubing $\mathrm{H}_{\mathrm{vL}}$

The loss level $H_{v L}$ is determined from chart 2. To do so the intersection point of the pumping stream Q $=20,0 \mathrm{~m}^{3} / \mathrm{h}$ with the pressure line DN 100 is searched. Starting with this intersection point a horizontal line is drawn on the side edge of the chart. The loss factor $\mathrm{H}_{\mathrm{vL}}$ for 100 m tubing can now be read off from here.
$\mathrm{H}_{\mathrm{vL}} 100=0,70 \mathrm{~m} / 100 \mathrm{~m}$ tubing
The total loss factor for the tubing results from the multiplication with the tubing length $L_{D}$.

| $\mathrm{H}_{\mathrm{VL}}$ | $=\mathrm{H}_{\mathrm{VL} 100} \cdot \mathrm{~L}_{\mathrm{D}}$ |
| ---: | :--- |
|  | $=\frac{0,7 \mathrm{~m}}{100 \mathrm{~m}} \cdot 25 \mathrm{~m}$ |
| $\mathrm{H}_{\mathrm{VL}}$ | $=0,18 \mathrm{~m}$ |

## 4. Determination of the loss factor $\mathrm{H}_{\mathrm{vE}}$ of the mounting parts and fittings

In table 6 and chart 3 the zeta values for fittings and mouldings can be determined. The following fittings and mouldings are supposed to be installed in the pressure line:

| piece | name | $\zeta$ | for $Q=20 \mathrm{~m}^{3} / \mathrm{h}$ |
| :--- | :--- | :--- | :--- |
| 1 | sluice valve DN 100 | 0,34 | 0,34 |
| 3 | elbows DN 100, $90^{\circ}$ | 0,35 | 1,05 |
| 1 | swing-type check valve R 101 | 7,00 | 7,00 |

## 5. Total loss factor $\mathrm{H}_{\mathrm{v}}$

The total loss factor results from the addition of all single loss factors

| $H_{v}$ | $=H_{v L}+H_{v E}$ |
| ---: | :--- |
|  | $=0,18 \mathrm{~m}+0,2 \mathrm{~m}$ |
|  | $=0,38 \mathrm{~m}$ |

## 6. Geodetic head $\mathrm{H}_{\text {geo }}$

The level difference between the switch-off point of the pump and the transfer point is called geodetic head. In this example the $\mathrm{H}_{\text {geo }}=3,1 \mathrm{~m}$.


## 7. Manometric head $\mathrm{H}_{\text {man }}$

The manometric head is the sum of the total loss factor and geodetic head lift.
$\begin{aligned} H_{\text {man }} & =H_{v}+H_{\text {geo }} \\ & =0,38 \mathrm{~m}+3,1 \mathrm{~m}=3,48 \mathrm{~m} \\ \mathrm{H}_{\text {man }} & \approx 3,5 \mathrm{~m}\end{aligned}$

## 8. System selection

The values $Q=20 \mathrm{~m}^{3} / \mathrm{h}$ (see pt. 2) and $\mathrm{H}_{\text {man }}=3,5 \mathrm{~m}$ (see pt. 7) yield the „desired operation point".
It is used to pre-dimension the sewage disposal unit.
The pump characteristic has to be above the desired operation point.
Since it is a boarding house where, according to DIN EN 12056-4 part 1, waste water discharge may not be interrupted, an automatic spare pump or a double unit has to be provided for. Therefore a double unit is selected for safety reasons.

Selected unit: compli 1010/4 BW


This way the unit can be sufficiently dimensioned. If, however, the precise operation point is required, the pipe or unit characteristic has to be determined and entered into the above chart.
To do so several pumping volumes are assumed at random. Then the corresponding loss factors are determined (see pt. 3-7).
The geodetic head $\mathrm{H}_{\text {geo }}$ is to be entered as a constant on the y -axis. The determined $\mathrm{H}_{\mathrm{v}}$ values are added to it.


In the example at hand the losses for the quantities $Q=30$ und $40 \mathrm{~m}^{3} / \mathrm{h}$ were determined and entered into the chart. Connecting the points found this way results in the unit or pipe characteristic. The intersection point yields the actual operation point of the pump.

The pump has a capacity $Q=32,0 \mathrm{~m}^{3} / \mathrm{h}$ bei $\mathrm{H}_{\text {man }}=3,8 \mathrm{~m}$.


## 9. Verification of the flow velocity v

The flow velocity should, in order to avoid, water hammers and impacts on the reflux valve, not be greater than $v_{\text {max }}=2,3 \mathrm{~m} / \mathrm{s}$ sein.
$v=\frac{Q_{p}}{V_{D / m}}=\frac{32 \mathrm{~m}^{3} / \mathrm{h}}{8,0 \mathrm{l} / \mathrm{m} \cdot 3,6}=\underline{\underline{1,11 \mathrm{~m} / \mathrm{s}}} \quad\left[3,6=\right.$ conversion factor $\left.\mathrm{m}^{3} / \mathrm{h} \mathrm{in} \mathrm{l} / \mathrm{s}\right]$
$v_{\text {min }} \leqslant v<v_{\text {max }}$
$\underline{\underline{0,7 \mathrm{~m} / \mathrm{s} \leq 1,11 \mathrm{~m} / \mathrm{s}<2,3 \mathrm{~m} / \mathrm{s}}} \Rightarrow$ The flow velocity is within the permitted range.

### 5.2 CALCULATION EXAMPLE 2

## Rainwater pumping unit with drain pumps UAK and concrete sump

Attention: In eplosion hazadous locations only the ioeration of ex-proof pumps is permitted.

## 1. Quantity determination according to DIN 1986 part 100

In table 4 the drainage coefficients $C$ depending on the kind of connected precipitation area can be determined.
The following precipitation a reas are connected

| Name | Area | Drainage coeffiecient |
| :--- | :--- | :--- |
| roof (slope $\geqslant 3^{\circ}$ ) | A1 $=170,0 \mathrm{~m}^{2}$ | $C=1,0$ |
| footpath with pavement $10 \times 10 \mathrm{~cm}$ | $A 2=110,0 \mathrm{~m}^{2}$ | $C=0,6$ |
| parking site with black top | $A 3=76,5 \mathrm{~m}^{2}$ | $C=1,0$ |

For the calculation it is necessary to have knowledge regarding the assessment level of rainfall which varies greatly from region to region. For a precise calculation the factor has to be requested with the local building authority. An overview can be taken from EN 12056-4 appendix A.

For the example at hand a mean assessment level of rainfall is assumed at


The conversion factor 3,6 can be used to convert from unit [ $\mathrm{l} / \mathrm{s}]$ to $\left[\mathrm{m}^{3} / \mathrm{h}\right]$.
$\Sigma V \cdot_{r 1-3}=Q \quad=\quad 22,5 \mathrm{~m}^{3} / \mathrm{h}$

## 2. Dimensionierung der Druckleitung

The unit is supposed to be connected via a pressure line of $L=520 \mathrm{~m}$. The operational roughness is $\mathrm{kb}=0,25 \mathrm{~mm}$.
The minimum flow velocity inside pressure lines is $\mathrm{v}_{\text {min }}>0,7 \mathrm{~m} / \mathrm{s}$.
In order to keep energy costs as low as possible it is attempted that the flow velocity does not rise considerably above $\mathrm{v}=1,0 \mathrm{~m} / \mathrm{s}$.

The determination of the necessary diameter can be performed with chart 2 . Therefore the chart is entered from top to bottom with $Q=22,5 \mathrm{~m}^{3} / \mathrm{h}$. When the quantity Q intersects the line $\mathrm{v}_{\text {min }}$ one can see that the intersection point is between the diagonal rows of diameters $\mathrm{d}_{\mathrm{i}} 100$ and di 125 . This means in the pressure line $\mathrm{d}_{\mathrm{i}} 100 \mathrm{v}$ is $>0,7 \mathrm{~m} / \mathrm{s}$ and in the line $\mathrm{d}_{\mathrm{i}} 125 \mathrm{v}$ is $<0,7 \mathrm{~m} / \mathrm{s}$.

Extending the vertical line further to the diagonal di 100 yields the result that the flow velocity of this line is approx. $v=0,8 \mathrm{~m} / \mathrm{s}$ and therewith greater than the minimum flow velocity of $\mathrm{V}_{\text {min }}=0,7 \mathrm{~m} / \mathrm{s}$ ist.

Chosen: pressure line DN 100

## 3. Determination of the pipe friction losses $\mathrm{H}_{\mathrm{vL}}$

The loss factor Hv is taken from chart 2 :
To do so the intersection point of the pumping capacity $Q=22,5 \mathrm{~m}^{3} / \mathrm{h}$ with the pressure line $\mathrm{d}_{\mathrm{i}} 100$ is determined. Starting from this intersection point a horizontal line is drawn on the side edge of the chart. The loss factor $\mathrm{H}_{\mathrm{vL} 100}$ for 100 m of pipe can be read off from here.
$\mathrm{H}_{\mathrm{vL} 100}=0,90 \mathrm{~m} / 100 \mathrm{~m}$ conduit
The total loss factor for the tubing is determined by multiplication with the pipe length L .

| $\mathrm{H}_{\mathrm{vL}}$ | $=\mathrm{H}_{\mathrm{vL} 100} \cdot \mathrm{~L}$ |
| :--- | :--- |
| $\mathrm{H}_{\mathrm{vL}}$ | $=\frac{0,9 \mathrm{~m}}{100 \mathrm{~m}} \cdot 520 \mathrm{~m}$ |
| $\mathrm{H}_{\mathrm{vL}}$ | $=4,68 \approx 4,7 \mathrm{~m}$ |

## 4. Determination of the friction losses $\mathrm{H}_{\mathrm{VE}}$ of the mounting parts and fittings

In table 6 and chart 3 the zeta values for fittings and mounting parts can be determined. The following fittings and mountings are supposed to be installed in the pressure line and the pump sump:

| piece | name | 3 for $\mathrm{Q}=22,5 \mathrm{~m}^{3} / \mathrm{h}$ |  |
| :---: | :---: | :---: | :---: |
| 1 | sluice valve DN 100 | 0,34 | 0,34 |
| 12 | elbows DN 100, $90^{\circ}$ | 0,35 | 4,20 |
| 1 | swing-type check valve R 100 G , weight on the outside | 20,00 | 20,00 |
|  |  | $\Sigma 弓=$ | 24,54 |

Chart 4 yields $H_{V E}=0,8 \mathrm{~m}$ for $Q=22,5 \mathrm{~m}^{3} / \mathrm{h}$ and $\Sigma \zeta=24,54$.

## 5. Total friction loss $\mathrm{H}_{\mathrm{v}}$

The friction loss results from the addition of all single loss factors

| $H_{v}$ | $=H_{v L}$ | $+H_{v E}$ |
| ---: | :--- | :--- |
|  | $=4,7 \mathrm{~m}$ | $+0,8 \mathrm{~m}$ |
| $\mathrm{H}_{v}$ | $=5,5 \mathrm{~m}$ |  |

## 6. Geodetic head $\mathrm{H}_{\text {geo }}$

The level difference between the switch-off point of the pump and the outflow into the gravity sewer is called geodetic head. In this example the $\mathrm{H}_{\text {geo }}=1,8 \mathrm{~m}$


## 7. Manometric head $\mathrm{H}_{\text {man }}$

The manometric head is the sum of the total friction losses and geodetic head.

| $\mathrm{H}_{\text {man }}$ | $=\mathrm{H}_{\mathrm{v}}+\mathrm{H}_{\text {geo }}$ |
| ---: | :--- |
|  | $=5,5 \mathrm{~m}+1,8 \mathrm{~m}$ |
|  | $\underline{\underline{\mathrm{H}_{\text {man }}}}=$ |
|  | $=7,3 \mathrm{~m}$ |

## 8. Pump selection

The values $Q=22,5 \mathrm{~m}^{3} / \mathrm{h}$ (see pt. 2) and $\mathrm{H}_{\text {man }}=7,3 \mathrm{~m}$ (see pt. 7) yield the "desired operation point".
This is used to pre-dimension the pumps.
The pump characteristic has to be located above the desired operation point. Depending on the desired security the pump can be more or less overdimensioned.

Selected pump: UAK 25/4 CW 1


This way the unit can be sufficiently dimensioned. If, however, the precise operation point is required, the pipe and unit characteristic has to be determined and entered into the above chart.

To do so several pumping volumes are taken at random. Then the corresponding loss factors (s. pt. 3-7) are determined.

The geodetic head $\mathrm{H}_{\text {geo }}$ has to be set to the y -axis as constant. The determined Hv values are added on it.
Since it is a double unit it can also be determined how much water is pumped when both pumps are simultaneously working at peak load.

To do so the characteristic of the second pump is graphically added to the first characteristic. Then the actual operation points are determined.


In the example at hand the losses for the quantities $Q=30$ and $40 \mathrm{~m}^{3} / \mathrm{h}$ were determined and entered into the chart.
Connecting the points found this way results in the unit and pipe characteristic. The points of intersection indicate the actual operation points of the pumps for base and peak load.

The pumps have a performance of $Q=24,0 \mathrm{~m}^{3} / \mathrm{h}$ bei $\mathrm{H}_{\text {man }}=7,9 \mathrm{~m}$ (base load) and $Q=26,0 \mathrm{~m}^{3} / \mathrm{h}$ bei $\mathrm{H}_{\text {man }}=8,8 \mathrm{~m}$ (peak load).

Since pumps are usually set up in such a way that one pump can handle the total amount of occurring water the second pump is referred to as stand by pump. Only in special cases the so called peak load is used.


## 9. Checking of flow velocity v

The flow velocity should not exceed $\mathrm{v}_{\max }=2,3 \mathrm{~m}$ in order to avoid water hammers and impacts on the reflux valve.


## 10. Duration of switching period $\mathrm{T}_{\text {sp }}$

The chosen pumps of type UAK 25/4 CW1 are having a power consumption of P1 = 2,7 kW.
According to table 9 this results in a duration of switching period of
$\mathrm{T}_{\mathrm{Sp}} \quad=\quad 120 \mathrm{~s}$

## 11. Pump volume $V_{p}$

For the required minimum pump volume $\mathrm{V}_{\mathrm{p}}$ the following formula applies:


## 12. Sump selection

A sump KS 1500-D 100 is selected. The minimum hysteresis $h_{p}$ can be taken from chart 14 .
It is $V_{p}=46,9 \mathrm{l}$
$h_{p}=250 \mathrm{~mm}$ is the lowest hysteresis.
The volume $\mathrm{V}_{\text {su }}$ remaining inside the sump and the switch-off level $h_{\text {Aus }}$ could be determined in tabele 10.
$\mathrm{h}_{\text {Aus }}=250 \mathrm{~mm}$
The damming height $h$ results from
$h \quad=h_{p}+h_{\text {Aus }}$
$\mathrm{h}=250 \mathrm{~mm}+250 \mathrm{~mm}$
$\mathrm{h}=500 \mathrm{~mm}$

The sump volume results in
$\mathrm{V}_{\mathrm{su}}=310 \mathrm{l}$

## 6. PUMP DIMENSIONING SUPPORT

| Company: |  |  |
| :--- | :--- | :--- | :--- | :--- |
| Adress: | Phone/Fax: |  |
| ZIP, Place: | Project, Place: |  |

## Please send back by fax at +495204 / 80368 or by e-mail at info®jung-pumpen.de

| 1.0 Objects vor dewatering |  |
| :--- | :--- |
| Celler / single bath / washroom | $\square$ |
| Single appartment | $\square$ |
| Detached house | $\square$ |
| Two-/Multi-family house | $\square$ |
| Commercial operation / Office building | $\square$ |
| Guesthouse / Restaurant / Hotel | $\square$ |
| Industrial facility | $\square$ |
| Public building | $\square$ |
| Recreational facility / Sports venue | $\square$ |
| Initiate in public sewer system | $\square$ |

Other $\qquad$
1.1 Objects vor dewatering (DUs) Designation

Number
Urinal (booth or row, per position)
Single urinal
Washbasin, bidet
Shower without plug
Shower with plug, bathtub
Kitchen sink, dishwasher
Washing-machine 6 kg, floor drain DN 50
Washing-machine 12 kg , floor drain DN 70
Floor drain DN 100
WC 6 l, WC 7,5 I
WC 9 I

|  |
| :--- |
|  |
|  |
|  |
|  |
|  |
|  |
|  |

Other $\qquad$

| 1.2 Installation site of the pump/lifting station |  |
| :--- | :---: |
| Inside the building, above floor | $\square$ |
| Inside the building, underfloor | $\square$ |
| Outside the building, underfloor | $\square$ |
| Inlet depth (OKG - pipe bottom) | $\square$ |
| Load on the manhole cover | $\square$ |
| - Walkable | $\square$ |
| - Drivable for cars | $\square$ |
| - Drivable for trucks | $\square$ |
| Duct existing, $\emptyset$ | $\square$ |

Contact person
E-mail
Phone for questions

Stamp, signature

## 7. DESIGN HELP FOR CONTROL TECHNOLOGY


4. Additional features

$\square$ Strobe light:
$\square 230 \mathrm{~V}$ $\square 12 \mathrm{~V}$
$\square$ Profile half-cylinder
$\square$ Outside cabinet lighting
$\square$ Horn:
$\square 230 \mathrm{~V}$
$\square 12 \mathrm{~V}$
6. Breeze
Pump sump ventilation
$\square$ Pressure pipe ventilating
$\square$ Pressure pipe flushing number of strands: $\square 1 \square 2 \square 3 \square 4$
7. Other comments:
$\qquad$
$\qquad$

Contact person $\qquad$ Further questions (phone) $\qquad$ Date, stamp, signature $\qquad$


Sewer level below the basement floor


Sewer level above the basement floor

## 8. BACKPRESSURE LEVEL

In accordance with EN 12056-4 the top level of the street at the connection site is regarded as backpressure level if not otherwise defined by local regulations. Drainage objects below backpressure level have to be connected to the public sewage system by way of a pumping station with backpressure loop.

All drainage objects located above backpressure level are to be drained using natural incline. In case of a "backpressure situation" within the sewage system and the building connection lines the use of the connected drainage objects is possible through an automatically working wastewater pumping station.

## 9. FORMULA SYMBOLS USED

| Symbol | Explanation | Unit |
| :---: | :---: | :---: |
| A | Rainfall area | $\mathrm{m}^{2}$ |
| C | Run-off coefficient | - |
| DU | Connection values (design unit) | 1/s |
| DN | Nominal width | mm |
| $\mathrm{d}_{\mathrm{i}}$ | Inner diameter of conduit | mm |
| E | Resident | - |
| h | Banking level in the sump well | mm |
| $\mathrm{h}_{\text {Aus }}$ | Switch-off level in the sump well | mm |
| $\mathrm{h}_{\mathrm{p}}$ | Differential height | mm |
| $\mathrm{H}_{\text {geo }}$ | Geodetic head | m |
| $\mathrm{H}_{\text {man }}$ | Manometric head | m |
| $\mathrm{H}_{\mathrm{p}}$ | Pumping head at operation point | m |
| $\mathrm{H}_{\mathrm{v}}$ | Total friction losses | m |
| $\underline{\mathrm{H}_{\mathrm{VE}}}$ | Friction losses of fittings etc. | m |
| $\underline{\mathrm{H}_{\mathrm{vL}}}$ | Pipe friction losses | m |
| $\mathrm{H}_{\mathrm{v}, \mathrm{i}}$ | Pressure losses | m |
| K | Abflusskennzahl | - |
| k | Operational roughness | mm |
| $\mathrm{L}_{\mathrm{D}}$ | Length of pressure line | m |
| P | Point | - |
| $\mathrm{P}_{1}$ | Power consumption | kW |
| Q | Capacity | l/s |
| $\mathrm{Q}_{\mathrm{c}}$ | Continuous discharge | l/s |
| $\mathrm{Q}_{\mathrm{f}}$ | Infiltration water | l/s |
| $\mathrm{Q}_{\mathrm{g}}$ | Commercial and industrial waste water | l/s |
| $\mathrm{Q}_{\mathrm{h}}$ | Domestic waste water | l/s |
| $\mathrm{Q}_{\text {max }}$ | admissible waste water discharge for $\mathrm{v}_{\text {max }}=2,5 \mathrm{~m} / \mathrm{s}$ | l/s |
| $\mathrm{Q}_{\text {min }}$ | admissible waste water discharge for $\mathrm{v}_{\text {min }}=0,7 \mathrm{~m} / \mathrm{s}$ | 1/s |
| $\mathrm{Q}_{\mathrm{p}}$ | pump capacity at the operating point | $\mathrm{m}^{3} / \mathrm{h}$ oder l/s |
| $\mathrm{Q}_{\mathrm{R}}$ | Rainwater discharge | l/s |
| $\mathrm{Q}_{\mathrm{t}}$ | Dry weather discharge | l/s |
| $\mathrm{Q}_{\text {tot }}$ | Total waste water discharge | l/s |
| $\mathrm{Q}_{\mathrm{ww}}$ | Waste water discharge | l/s |
| Q | Inflow | m $3 / \mathrm{h}$ oder l/s |
| $\mathrm{q}_{\mathrm{n}}$ | Specific occurrence of domestic waste water | l/(s •1000 E) |
| $r^{5 / 2}$ | Five minute rain, once in two years | l/(s - ha) |
| $r^{5 / 100}$ | Five minute rain, once in 100 years | l/(s •ha) |
| $\mathrm{r}_{(0, \mathrm{~T})}$ | Assessment rainfall | l/(s • ha) |
| $\mathrm{T}_{\text {sp }}$ | Duration of switching period | 5 |
| v | Flow velocity | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{V}_{\mathrm{D}}$ | Volume of pipeline | 1 |
| $\mathrm{V}_{\mathrm{D} / \mathrm{m}}$ | Pipeline volume per meter | 1/m |
| $v_{\text {max }}$ | Maximum admissible flow velocity | $\mathrm{m} / \mathrm{s}$ |
| $v_{\text {min }}$ | Minimum flow velocity | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{V}_{\mathrm{p}}$ | Pumping volume | 1 |
| $\mathrm{V}_{\text {Su }}$ | Sump volume | - |
| v | Kinematic viscosity ( Ny ) | $\mathrm{mm}^{2} / \mathrm{s}$ |
| 3 | Drag coefficient (Zeta) | - |

Symbols used
10. PEHD PRESSURE LINES (EXCERPT)
7 LO8 NIO səd!d-aHヨd

| PEHD-pipes DIN 8074 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | PN 12,5-PE $80-$ SDR 11 |  |  |  |  |  | S 8 (PN 16) PE 100-SDR 17 |  |  |  |  |  | S 5 (PN 10) PE 100 -SDR 11 |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| DN | D | x | s | di | $V_{D / M}\left(V_{L}\right)$ | $\begin{gathered} Q_{\text {min }} \text { for } \\ v=0,7 \mathrm{~m} / \mathrm{s} \end{gathered}$ | D | x | S | di | $V_{D / M}\left(V_{L}\right)$ | $\begin{gathered} Q_{\text {min }} \text { for } \\ v=0,7 \mathrm{~m} / \mathrm{s} \end{gathered}$ | D | x | S | di | $V_{\text {D/M }}\left(V_{L}\right)$ | $\begin{gathered} Q_{\min } \text { for } \\ v=0,7 \mathrm{~m} / \mathrm{s} \end{gathered}$ |
| 20 | 25,0 | x | 2,3 | 20,4 | 0,33 | 0,23 | 25,0 | x | 1,8 | 21,4 | 0,36 | 0,25 |  |  |  |  |  |  |
| 25 | 32,0 | x | 2,9 | 26,2 | 0,54 | 0,38 | 32,0 | x | 1,9 | 28,2 | 0,62 | 0,44 | 32,0 | $x$ | 2,9 | 26,2 | 0,54 | 0,38 |
| 32 | 40,0 | x | 3,7 | 32,6 | 0,83 | 0,58 | 40,0 | x | 2,4 | 35,2 | 0,97 | 0,68 | 40,0 | x | 3,7 | 32,6 | 0,83 | 0,58 |
| 40 | 50,0 | x | 4,6 | 40,8 | 1,31 | 0,92 | 50,0 | x | 3,0 | 44,0 | 1,52 | 1,06 | 50,0 | x | 4,6 | 40,8 | 1,31 | 0,92 |
| 50 | 63,0 | x | 5,8 | 51,4 | 2,07 | 1,45 | 63,0 | $x$ | 3,8 | 55,4 | 2,41 | 1,69 | 63,0 | x | 5,8 | 51,4 | 2,07 | 1,45 |
| 65 | 75,0 | x | 6,8 | 61,4 | 2,96 | 2,07 | 75,0 | x | 4,5 | 66,0 | 3,42 | 2,39 | 75,0 | x | 6,8 | 61,4 | 2,96 | 2,07 |
| 80 | 90,0 | x | 8,2 | 73,6 | 4,25 | 2,98 | 90,0 | x | 5,4 | 79,2 | 1,93 | 3,45 | 90,0 | x | 8,2 | 73,6 | 4,25 | 2,98 |
| 100 | 110,0 | x | 10,0 | 90,0 | 6,36 | 4,45 | 110,0 | x | 6,6 | 96,8 | 7,36 | 5,15 | 110,0 | x | 10,0 | 90,0 | 6,36 | 4,45 |
|  | 125,0 | $x$ | 11,4 | 102,2 | 8,20 | 5,74 | 125,0 | x | 7.4 | 110,2 | 9,54 | 6,68 | 125,0 | x | 11,4 | 102,2 | 8,20 | 5,74 |
| 125 | 140,0 | $x$ | 12,8 | 114,4 | 10,28 | 7,20 | 140,0 | x | 8,3 | 123,4 | 11,96 | 8,37 | 140,0 | x | 12,7 | 114,6 | 10,31 | 7,22 |
| 150 | 160,0 | x | 14,6 | 130,8 | 13,44 | 9,41 | 160,0 | x | 9,5 | 141,0 | 15,61 | 10,93 | 160,0 | x | 14,6 | 130,8 | 13,44 | 9,41 |
|  | 180,0 | X | 16,4 | 147,2 | 17,02 | 11,91 | 180,0 | x | 10,7 | 158,6 | 19,76 | 13,83 | 180,0 | x | 16,4 | 147,2 | 17,02 | 11,91 |
|  | 200,0 | x | 18,2 | 163,6 | 21,02 | 14,71 | 200,0 | x | 11,9 | 176,2 | 24,38 | 17,07 | 200,0 | x | 18,2 | 163,6 | 21,02 | 14,71 |
| 200 | 225,0 | x | 20,5 | 184,0 | 26,59 | 18,61 | 225,0 | x | 13,4 | 198,2 | 30,85 | 21,60 | 225,0 | x | 20,5 | 184,0 | 26,59 | 18,61 |
|  | 250,0 | x | 22,8 | 204,4 | 32,81 | 22,97 | 250,0 | x | 14,8 | 220,4 | 38,15 | 26,71 | 250,0 | x | 22,7 | 204,6 | 32,88 | 23,02 |
| 250 | 280,0 | x | 25,5 | 229,0 | 41,19 | 28,83 | 280,0 | x | 16,6 | 246,8 | 47,84 | 33,49 | 280,0 | x | 25,4 | 229,2 | 41,26 | 28,88 |
|  | 315,0 | x | 28,7 | 257,6 | 52,12 | 36,48 | 315,0 | x | 18,7 | 277,6 | 60,52 | 42,37 | 315,0 | x | 28,6 | 257,8 | 52,20 | 36,54 |
| 300 | 355,0 | $x$ | 32,3 | 290,4 | 66,23 | 46,36 | 355,0 | x | 21,1 | 312,8 | 76,85 | 53,79 | 355,0 | x | 32,2 | 290,6 | 66,33 | 46,43 |
| 350 | 400,0 | x | 36,4 | 327,2 | 84,08 | 58,86 | 400,0 | x | 23,7 | 352,6 | 97,65 | 68,35 | 400,0 | x | 36,3 | 327,4 | 84,19 | 58,93 |
| 400 | 450,0 | x | 41,0 | 368,0 | 106,36 | 74.45 | 450,0 | x | 26,7 | 396,6 | 123,54 | 86,48 | 450,0 | x | 40,9 | 368,2 | 106,48 | 74,54 |

## JUNG PUMPEN

GERMANY: JUNG PUMPEN GmbH • Industriestr. 4-6 • 33803 Steinhagen • Phone +495204 170 • Fax +49520480368
infoßjung-pumpen.de • www.jung-pumpen.de
AUSTRIA: JUNG PUMPEN • Brown-Boveri-Str. 6/14 • 2351 Wiener Neudorf • Phone +43 2236866896
Fax +43 2236866930 - infoajung-pumpen.at • www.jung-pumpen.at


[^0]:    Source: EN 12056-2:2000, abstract from table 2

[^1]:    Source: DIN 1986-100, table 6

